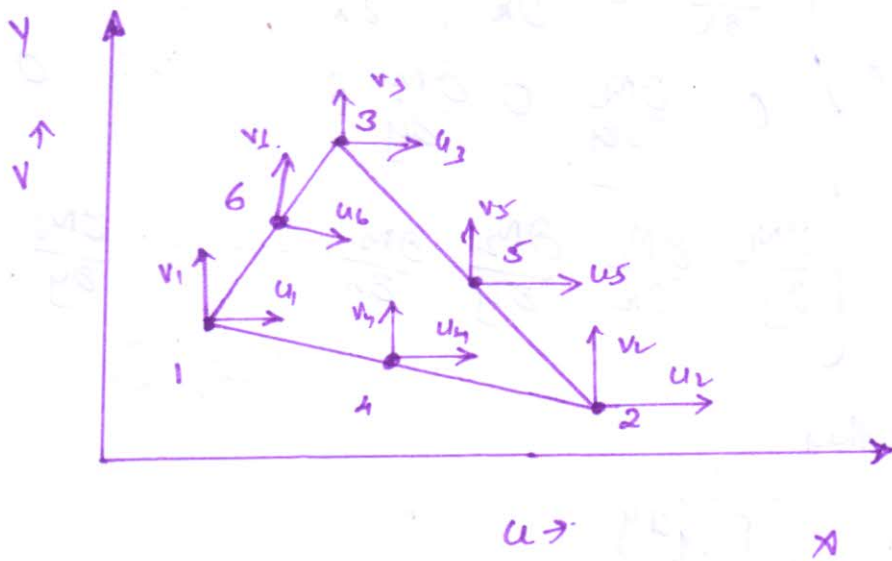


LST (Linear Strain Triangular) ELEMENT ⁽⁴⁾



A Six Noded triangular element is known as LST Element

12 unknown D.O.F. The displacement func of an element is quadratic. instead of linear as in the CST.

$$u = \{u, v\} = \{u_1, v_1, u_2, v_2, u_3, v_3, u_4, v_4, u_5, v_5, u_6, v_6\}^T$$

$$u = N_1 u_1 + N_2 u_2 + N_3 u_3 + N_4 u_4 + N_5 u_5 + N_6 u_6$$

$$v = N_1 v_1 + N_2 v_2 + N_3 v_3 + N_4 v_4 + N_5 v_5 + N_6 v_6$$

$$u = \sum_{i=1}^6 N_i u_i$$

$$v = \sum_{i=1}^6 N_i v_i$$

We know that the Shape functions.

$$N_1 = (2b_1 - 1) L_1$$

$$N_4 = 4L_1 L_2$$

$$N_2 = b_2 (2L_2 - 1)$$

$$N_5 = 4L_2 L_3$$

$$N_3 = b_3 (2L_3 - 1)$$

$$N_6 = 4L_1 L_3$$

$$\begin{Bmatrix} e_{xx} \\ e_{yy} \\ \gamma_{xy} \end{Bmatrix} = \begin{bmatrix} \frac{\partial N_1}{\partial x} & 0 & \frac{\partial N_2}{\partial x} & 0 & \frac{\partial N_3}{\partial x} & 0 & \dots & \frac{\partial N_6}{\partial x} & 0 \\ 0 & \frac{\partial N_1}{\partial y} & 0 & \frac{\partial N_2}{\partial y} & 0 & \dots & 0 & \frac{\partial N_6}{\partial y} \\ \frac{\partial N_1}{\partial y} & \frac{\partial N_1}{\partial x} & \frac{\partial N_2}{\partial y} & \frac{\partial N_2}{\partial x} & \dots & \frac{\partial N_6}{\partial y} & \frac{\partial N_6}{\partial x} \end{bmatrix} \begin{Bmatrix} u_1 \\ v_1 \\ u_2 \\ v_2 \\ \vdots \\ u_6 \\ v_6 \end{Bmatrix}$$

→ ③

We know that

$$\{e\} = [B]\{u\}$$

The informa needed to express deriv of shape func. w.r.t. global co-ordinates $x+y$ is given below in form

$$\frac{\partial N_1}{\partial x} = \frac{\partial N_1}{\partial L_1} \cdot \frac{\partial L_1}{\partial x} + \frac{\partial N_1}{\partial L_2} \cdot \frac{\partial L_2}{\partial x} + \frac{\partial N_1}{\partial L_3} \cdot \frac{\partial L_3}{\partial x}$$

$$\frac{\partial N_1}{\partial y} = \frac{\partial N_1}{\partial L_1} \cdot \frac{\partial L_1}{\partial y} + \frac{\partial N_1}{\partial L_2} \cdot \frac{\partial L_2}{\partial y} + \frac{\partial N_1}{\partial L_3} \cdot \frac{\partial L_3}{\partial y}$$

Atla Co-ordinates are expressed as

$$L_i = a_i + b_i x + c_i y$$

$$a_i = \frac{(x_j y_k - y_j x_k)}{2A}$$

$$b_i = \frac{(y_j - y_k)}{2A}$$

$$c_i = \frac{(x_k - x_j)}{2A}$$

④

⑤

(Note the QA is included in the defn of coeff. ⁶

$$\frac{\partial b_1}{\partial x} = b_1 \quad \frac{\partial b_2}{\partial x} = b_2 \quad \frac{\partial b_3}{\partial x} = b_3 \quad \left. \vphantom{\frac{\partial b_1}{\partial x}} \right\} \textcircled{6}$$

$$\frac{\partial b_1}{\partial y} = c_1 \quad \frac{\partial b_2}{\partial y} = c_2 \quad \frac{\partial b_3}{\partial y} = c_3 \quad \left. \vphantom{\frac{\partial b_1}{\partial y}} \right\}$$

Using Lem. $\textcircled{2}$, $\textcircled{4}$ + $\textcircled{6}$.

We know then

$$\frac{\partial N_1}{\partial x} = \frac{\partial N_1}{\partial x} (2b_1 - 1)$$

$$\frac{\partial N_1}{\partial x} = \frac{\partial N_1}{\partial b_1} \cdot \frac{\partial b_1}{\partial x} + \frac{\partial N_1}{\partial b_2} \cdot \frac{\partial b_2}{\partial x} + \frac{\partial N_1}{\partial b_3} \cdot \frac{\partial b_3}{\partial x}$$

$$= \frac{\partial N_1}{\partial b_1} (2b_1 - 1) \cdot b_1 + 0 + 0$$

$$= \frac{\partial}{\partial b_1} (2b_1^2 - b_1) b_1 = \underline{\underline{(4b_1 - 1) b_1}}$$

$$\boxed{\frac{\partial N_1}{\partial x} = b_1 (4b_1 - 1)}$$

III.

$$\frac{\partial N_1}{\partial x} = b_1 (4b_1 - 1)$$

$$\frac{\partial N_1}{\partial y} = c_1 (4b_1 - 1)$$

$$\frac{\partial N_2}{\partial x} = b_2 (4b_2 - 1)$$

$$\frac{\partial N_2}{\partial y} = c_2 (4b_2 - 1)$$

$$\frac{\partial N_3}{\partial x} = b_3 (4b_3 - 1)$$

$$\frac{\partial N_3}{\partial y} = c_3 (4b_3 - 1)$$

$$\frac{\partial N_4}{\partial x} = 4(l_2 b_1 + l_1 b_2), \quad \frac{\partial N_4}{\partial y} = 4(l_2 c_1 + l_1 c_2)$$

$$\frac{\partial N_5}{\partial x} = 4(l_3 b_2 + l_2 b_3), \quad \frac{\partial N_5}{\partial y} = 4(l_3 c_2 + l_2 c_3)$$

$$\frac{\partial N_6}{\partial x} = 4(l_1 b_3 + l_3 b_1), \quad \frac{\partial N_6}{\partial y} = 4(l_1 c_3 + l_3 c_1)$$

Stress are related in the of element as

$$\underline{\{\sigma\}} = [D] \underline{\{e\}}$$

Strain energy in the element is $U_e = \frac{1}{2} \int_V \underline{\{e\}}^T \underline{\{\sigma\}} \cdot dV$

$$U_e = \frac{1}{2} \{u\}^T \left[\int_V [B]^T [D] [B] \cdot dV \right] \{u\}$$

$$[K] = \left[\int_V [B]^T [D] [B] \cdot dV \right]$$