

# Strain Displacement Matrix [B] for CST. ELEMENT.

We know

$$\begin{aligned}
 u &= N_1 u_1 + N_2 u_2 + N_3 u_3 \\
 v &= N_1 v_1 + N_2 v_2 + N_3 v_3
 \end{aligned}$$

The Strain Components for CST elements are

Normal Strain  $\epsilon_x = \frac{du}{dx}$ ,  $\epsilon_y = \frac{dv}{dy}$ ,  $\gamma_{xy} = \frac{dv}{dx} + \frac{du}{dy}$

$$\epsilon_x = \frac{\partial N_1}{\partial x} u_1 + \frac{\partial N_2}{\partial x} u_2 + \frac{\partial N_3}{\partial x} u_3$$

$$\epsilon_y = \frac{\partial N_1}{\partial y} v_1 + \frac{\partial N_2}{\partial y} v_2 + \frac{\partial N_3}{\partial y} v_3$$

$$\gamma_{xy} = \frac{\partial N_1}{\partial y} u_1 + \frac{\partial N_2}{\partial y} u_2 + \frac{\partial N_3}{\partial y} u_3 + \frac{\partial N_1}{\partial x} v_1 + \frac{\partial N_2}{\partial x} v_2 + \frac{\partial N_3}{\partial x} v_3$$

Arranging the strain  $\epsilon_x, \epsilon_y, \gamma_{xy}$  in matrix

$$\begin{bmatrix} \epsilon_x \\ \epsilon_y \\ \gamma_{xy} \end{bmatrix} = \begin{bmatrix} \frac{\partial N_1}{\partial x} & 0 & \frac{\partial N_2}{\partial x} & 0 & \frac{\partial N_3}{\partial x} & 0 \\ 0 & \frac{\partial N_1}{\partial y} & 0 & \frac{\partial N_2}{\partial y} & 0 & \frac{\partial N_3}{\partial y} \\ \frac{\partial N_1}{\partial y} & \frac{\partial N_1}{\partial x} & \frac{\partial N_2}{\partial y} & \frac{\partial N_2}{\partial x} & \frac{\partial N_3}{\partial y} & \frac{\partial N_3}{\partial x} \end{bmatrix} \begin{bmatrix} u_1 \\ v_1 \\ u_2 \\ v_2 \\ u_3 \\ v_3 \end{bmatrix} \rightarrow \text{⑥}$$

$$N_1 = \frac{P_1 + Q_1 x + R_1 y}{2A} \Rightarrow \frac{\partial N_1}{\partial x} = \frac{Q_1}{2A} \quad \frac{\partial N_1}{\partial y} = \frac{R_1}{2A}$$

$$N_2 = \frac{P_2 + Q_2 x + R_2 y}{2A} \Rightarrow \frac{\partial N_2}{\partial x} = \frac{Q_2}{2A} \quad \frac{\partial N_2}{\partial y} = \frac{R_2}{2A}$$

$$N_3 = P_3 + q_3 x + r_3 y \quad \Rightarrow \quad \left. \frac{\partial N_3}{\partial x} = \frac{q_3}{2A}, \quad \frac{\partial N_3}{\partial y} = \frac{r_3}{2A} \right\} \rightarrow \textcircled{c}$$

Subst. Eq.  $\textcircled{c}$  in  $\textcircled{b}$   
we get

$$\begin{Bmatrix} e_x \\ e_y \\ \gamma_{xy} \end{Bmatrix} = \frac{1}{2A} \begin{bmatrix} q_1 & 0 & q_2 & 0 & q_3 & 0 \\ 0 & r_1 & 0 & r_2 & 0 & r_3 \\ r_1 & q_1 & r_2 & q_2 & r_3 & q_3 \end{bmatrix} \begin{Bmatrix} u_1 \\ v_1 \\ u_2 \\ v_2 \\ u_3 \\ v_3 \end{Bmatrix} \rightarrow \textcircled{d}$$

Which is of the form

$$\{e\} = [B] \{u\}$$

So

$$[B] = \frac{1}{2A} \begin{bmatrix} q_1 & 0 & q_2 & 0 & q_3 & 0 \\ 0 & r_1 & 0 & r_2 & 0 & r_3 \\ r_1 & q_1 & r_2 & q_2 & r_3 & q_3 \end{bmatrix}$$

When

$$q_1 = y_2 - y_3$$

$$r_1 = x_3 - x_2$$

$$q_2 = y_3 - y_1$$

$$r_2 = x_1 - x_3$$

$$q_3 = y_1 - y_2$$

$$r_3 = x_2 - x_1$$

# Stiffness Matrix Equation for 2D Element (CST.)

We know  $[K] = \int_V [B]^T [D] [B] dV$

$[K] \cdot [B]^T [D] [B] \cdot V = [B]^T [D] [B] = A \cdot t$

$[K] = [B]^T [D] [B] \cdot A \cdot t$

Area of triangle  $= \frac{1}{2} \begin{vmatrix} 1 & x_1 & y_1 \\ 1 & x_2 & y_2 \\ 1 & x_3 & y_3 \end{vmatrix}$

$t \rightarrow$  thickness of element

$[B] \rightarrow$  Strain Displacement matrix

$[B] = \frac{1}{2A} \begin{bmatrix} a_1 & 0 & a_2 & 0 & a_3 & 0 \\ 0 & \gamma_1 & 0 & \gamma_2 & 0 & \gamma_3 \\ \gamma_1 & a_1 & \gamma_2 & a_2 & \gamma_3 & a_3 \end{bmatrix}$

$[D] \rightarrow$  stress strain relationship matrix

$[D] = \frac{E}{1-\mu^2} \begin{bmatrix} 1 & \mu & 0 \\ \mu & 1 & 0 \\ 0 & 0 & \frac{1-\mu}{2} \end{bmatrix}$  (Plane Stress prob)

$[D] = \frac{E}{(1+\mu)(1-2\mu)} \begin{bmatrix} (1-\mu) & \mu & 0 \\ \mu & (1-\mu) & 0 \\ 0 & 0 & \frac{1-2\mu}{2} \end{bmatrix}$

(Plane Strain problem)

$E \rightarrow$  Young's modulus

$\mu \rightarrow$  Poisson's Ratio